

JAMS

# Analysis of Fastener Disbond Arrest Mechanism for Laminated Composite Structures

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The Joint Advanced Materials and Structures Center of Excellence



# FAA Sponsored Project Information



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- **Other FAA Personnel:** Larry Ilcewicz, Peter Shyprykevich (Ret.)
- **Industry Participants:** Gerald Mabson, Eric Cregger, Marc Piehl, Cliff Chen, Lyle Deobald, Alan Miller, Steve Precup (All from Boeing)
- **Industry Sponsors:** Boeing

## Work Accomplished: Phase 1

### (“Development of Reliability-Based Damage Tolerant Structural Design Methodology”)

- Developed the methodology to determine the reliability and maintenance planning of damage tolerant structures.
- Developed a user-friendly software (RELACS) for calculating POF and inspection intervals.
- Developed software interface (VSTM) with Nastran to facilitate stochastic FEA.
- Implemented stochastic FEA to obtain initial/damaged residual strength variance.

### Current Research

- Develop analytical methods to analyze disbond and delamination arrest mechanisms in bonded structures under mixed mode loading.
- To apply probabilistic methods to assess reliability of bonded structures with fasteners.

# Analysis of Disbond/Delamination Arrest Mechanisms

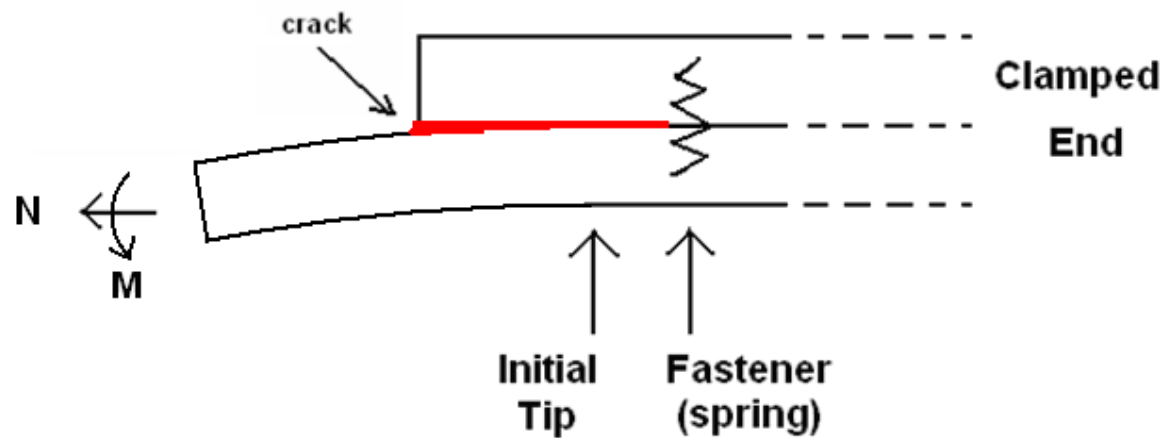
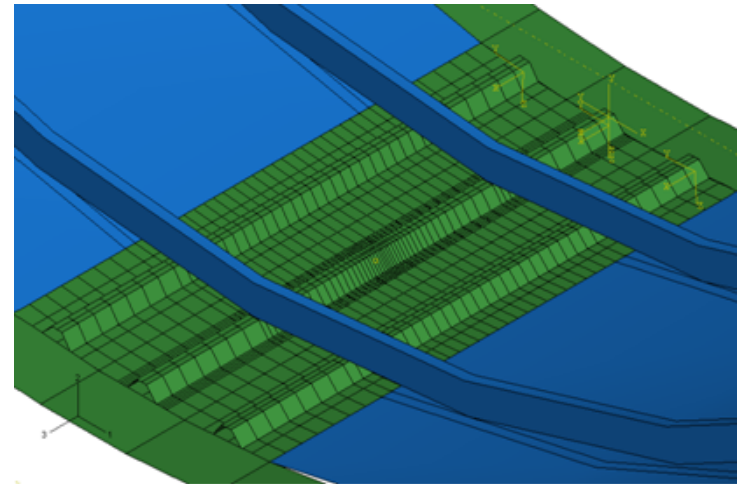
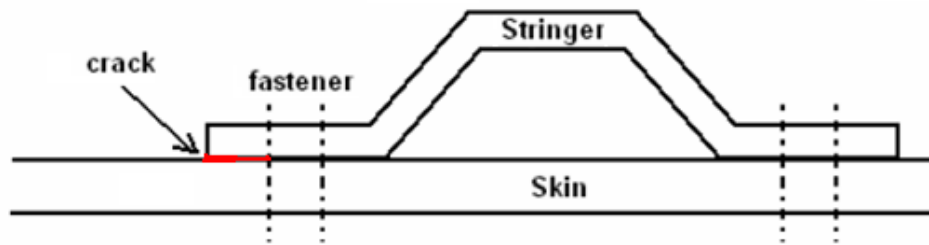
- **Objectives**

- To understand the effectiveness of delamination/disbond arrest mechanisms
- To develop analysis tools for design and optimization

- **Tasks**

- 1). Establish FE models in ABAQUS
- 2). Develop 1-D (beam) and 2D (plate) analytical capabilities
- 3). Implement reliability analysis capability
- 4). Conduct sensitivity studies on fastener effectiveness and stacking sequence effects

# JAMS Bonded Skin/Stiffener with Fasteners



# Laminate Configuration (16 plies)

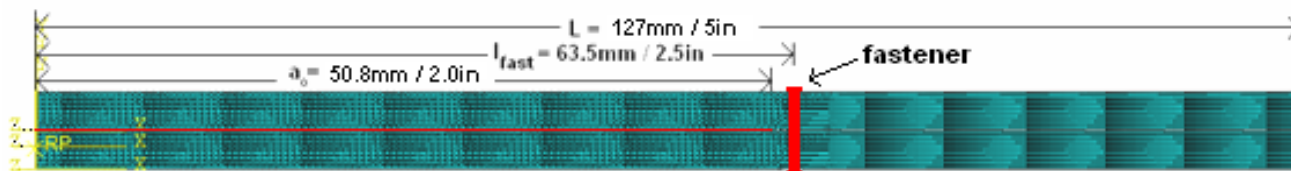
0-ply	Lay-up	$E_x$	C (in/lb) (joint compliance)
25.0%	$(45/0/-45/90/45/0/-45/90)_s$	$7.42 \times 10^6$	$7.73 \times 10^{-6}$
37.5%	$(45/0/-45/0/45/0/-45/90)_s$	$9.29 \times 10^6$	$6.57 \times 10^{-6}$
50.0%	$(45/0_2/-45/0_2/90_2)_s$	$1.10 \times 10^7$	$5.85 \times 10^{-6}$
62.5%	$(45/0_3/-45/0_2/90)_s$	$1.30 \times 10^7$	$5.25 \times 10^{-6}$

$$C = \left( \frac{t_1 + t_2}{2d} \right)^a \frac{b}{n} \left( \frac{1}{t_1 E_1} + \frac{1}{nt_2 E_2} + \frac{1}{nt_1 E_3} + \frac{1}{2nt_2 E_3} \right)$$

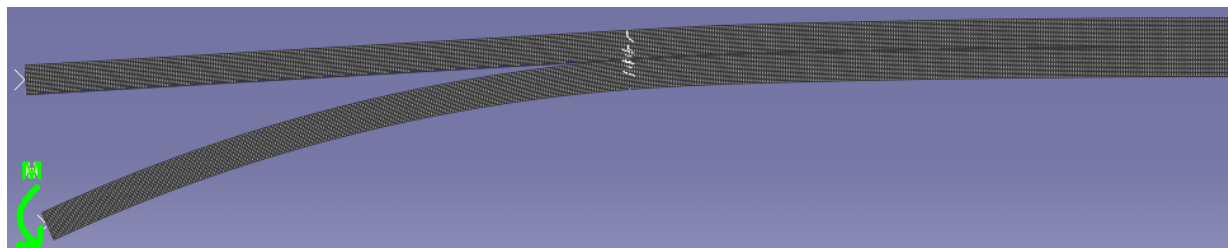
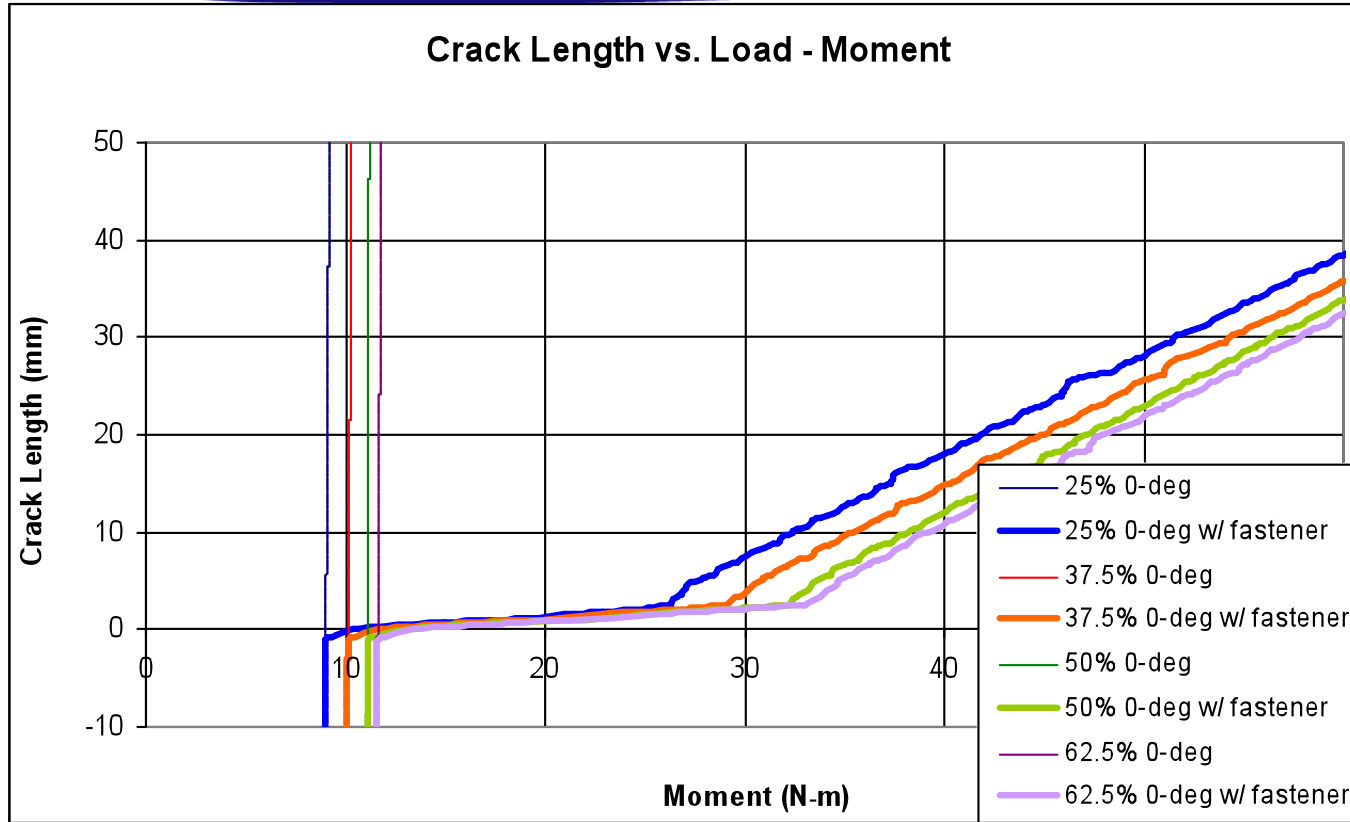
$$a = 2/3, \quad b = 4.2, \quad n = 1$$

$$k_{clamp} = \frac{AE}{(t_1 + t_2)} = 3.37 \times 10^6$$

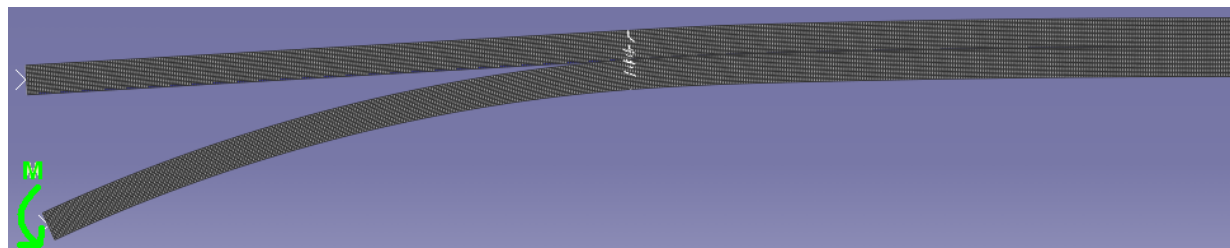
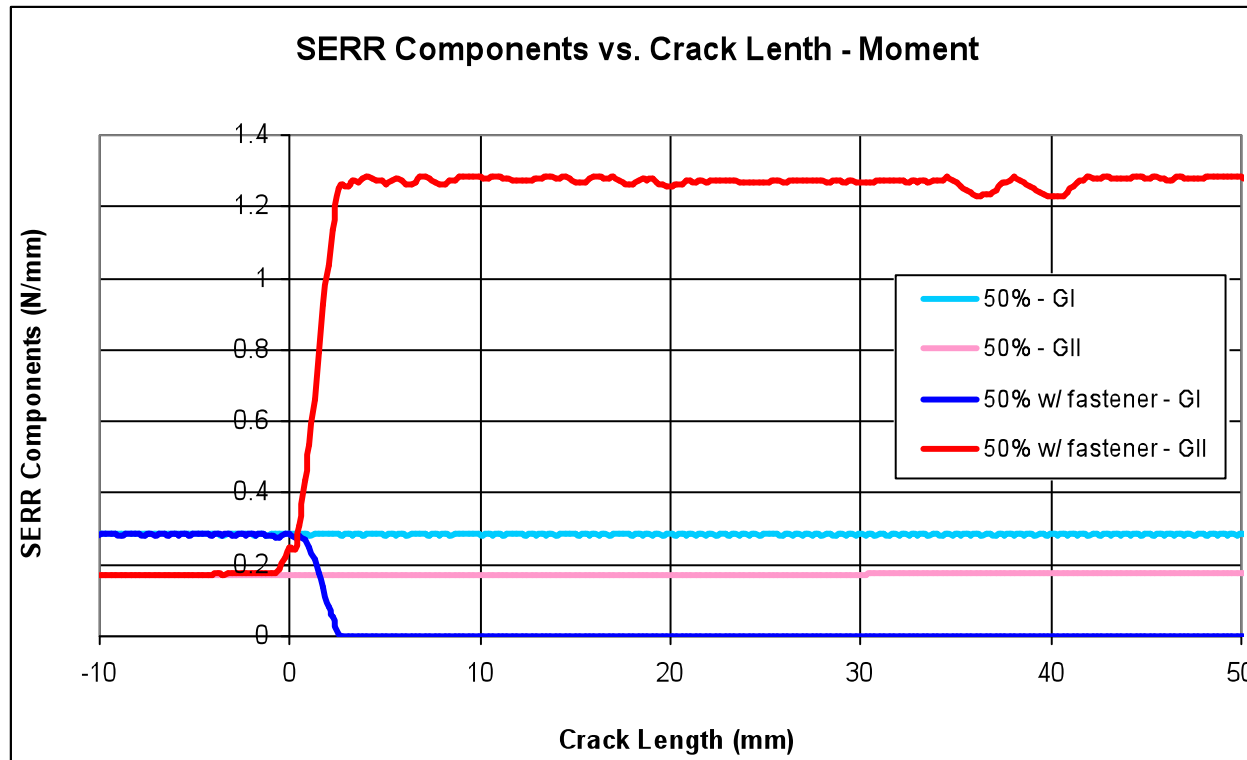
$$G_{equivC} = G_{IC} + (G_{IIC} - G_{IC}) \left( \frac{G_{II}}{G_I + G_{II}} \right)^n$$



# Results: Applied Moment Only

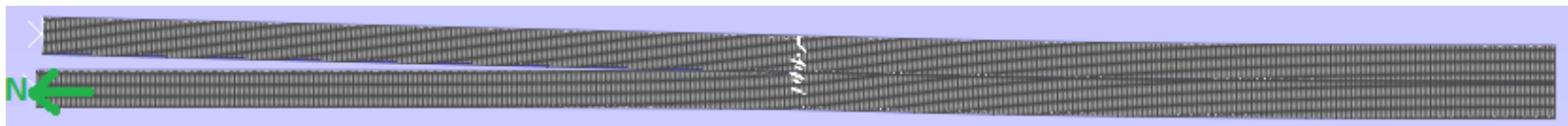
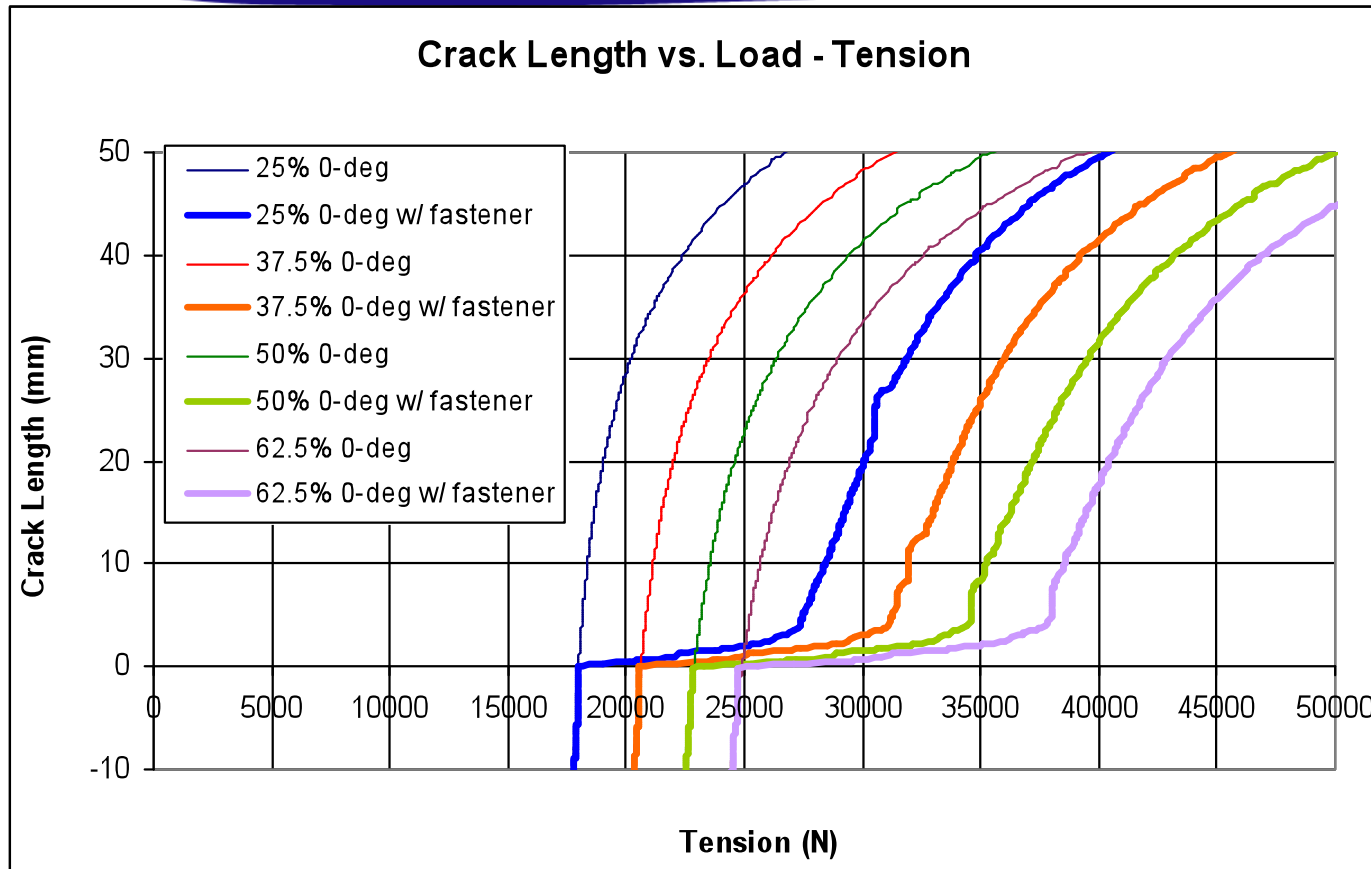


# Mode Decomposition with Fastener: Applied Moment Only

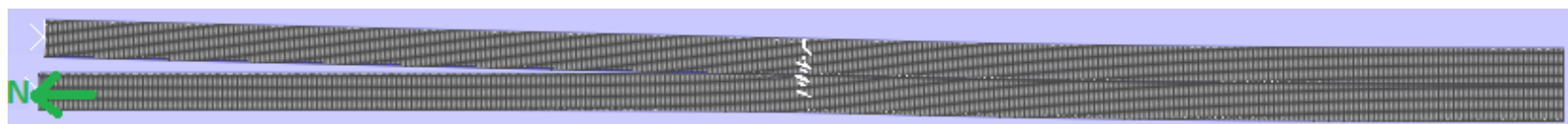
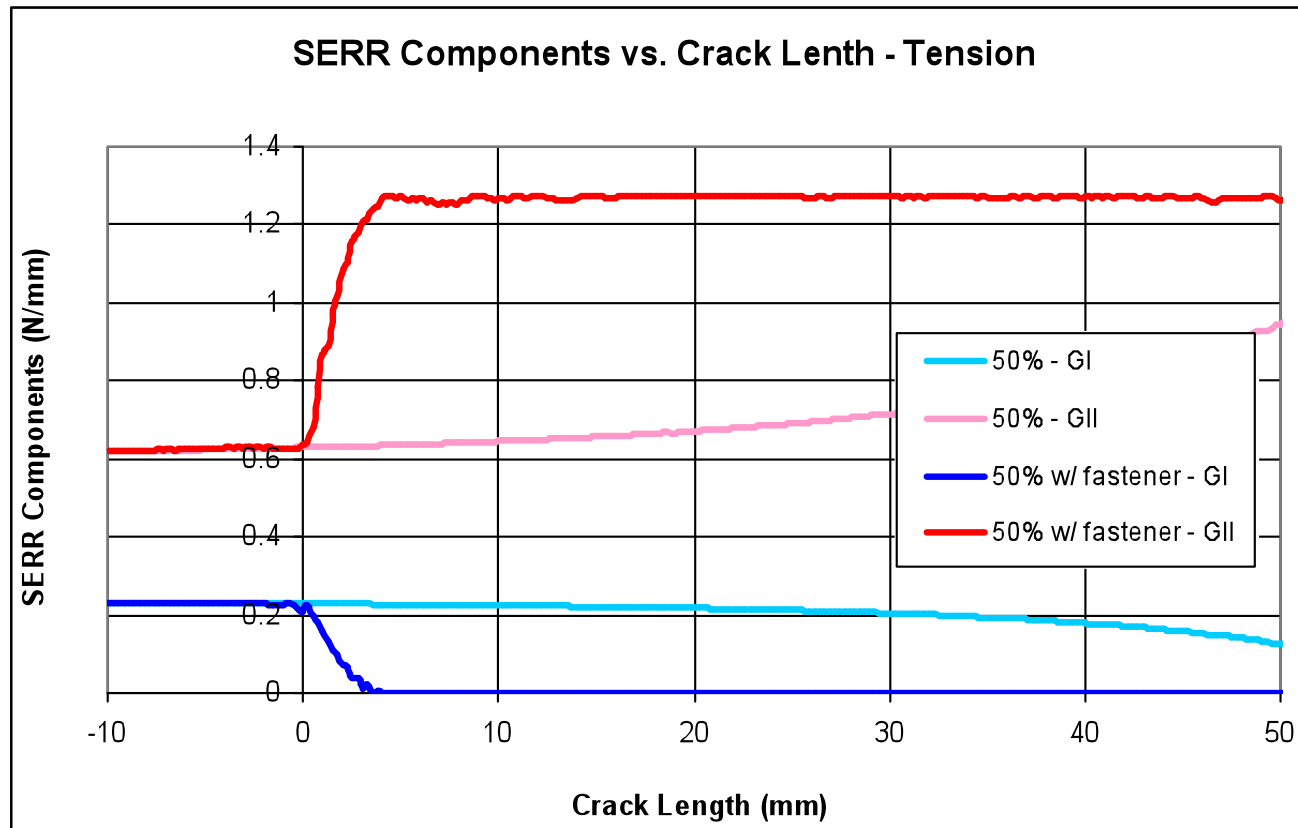




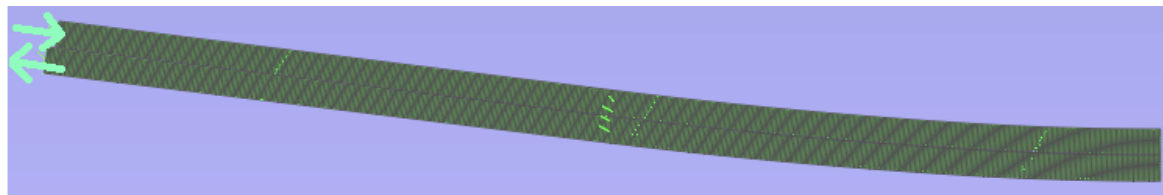
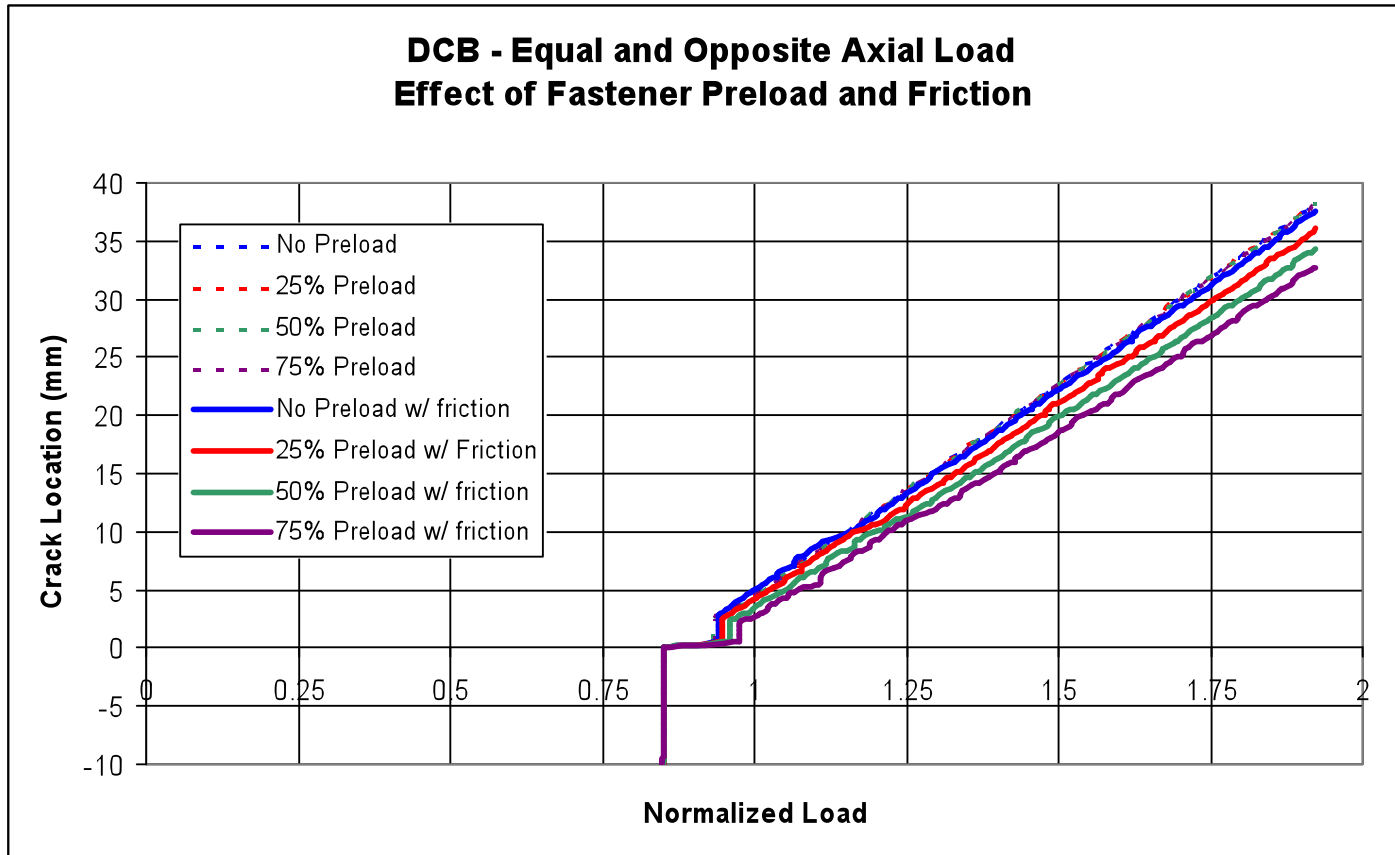
# Results: Applied Tension Only



# Mode Decomposition: Applied Tension Only

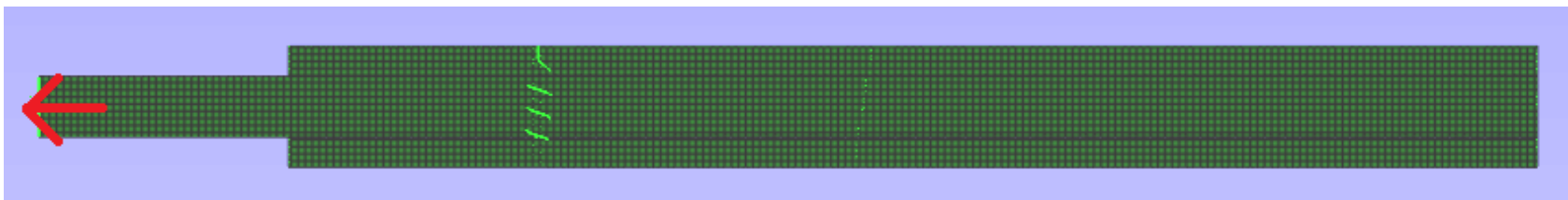


# Friction and Fastener Preload

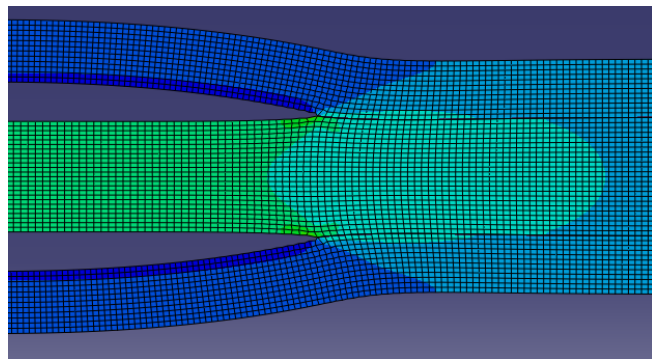
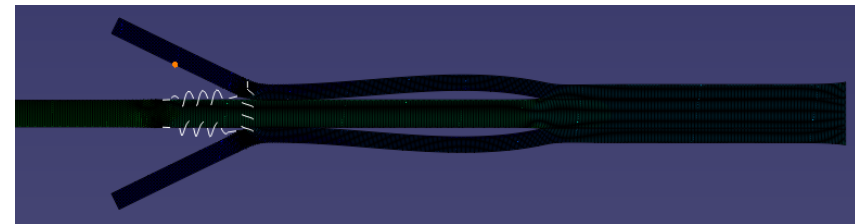
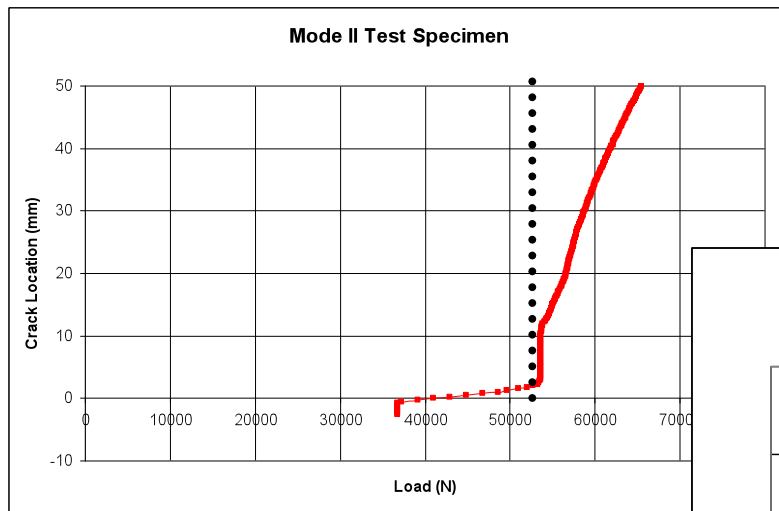


# Mode II Test Specimen

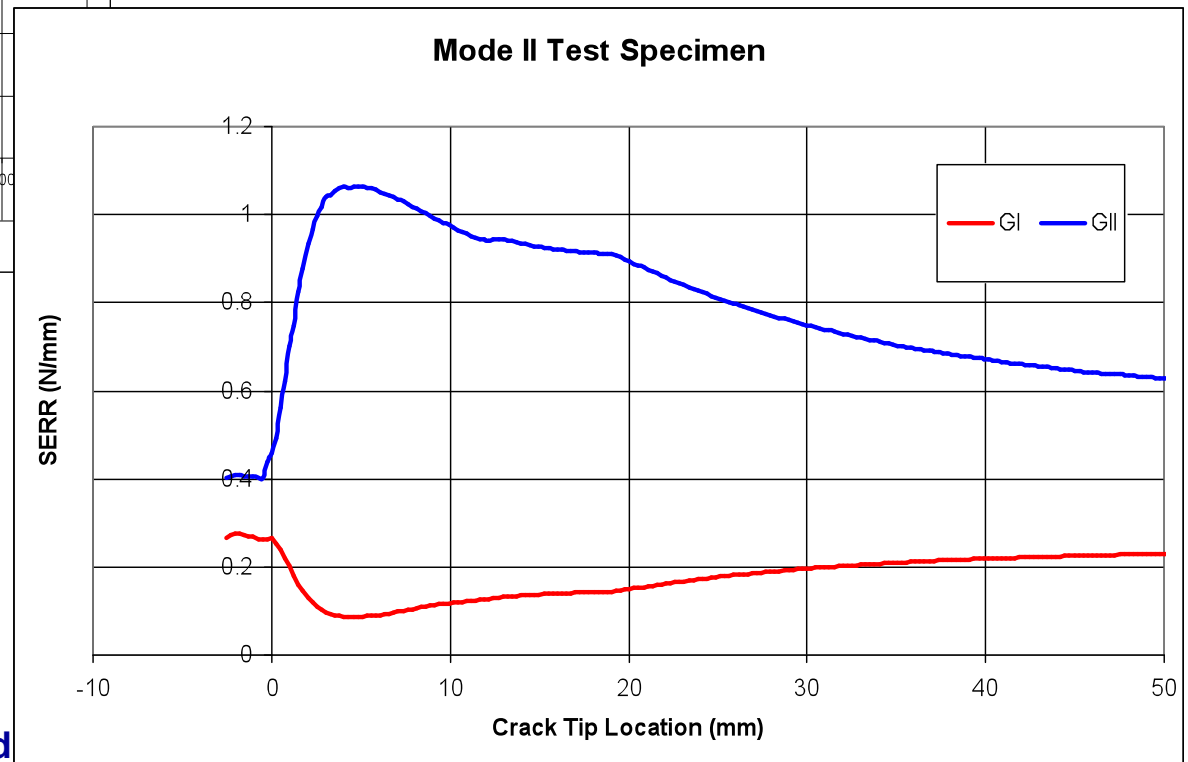
- Classical “bending type” specimen not suitable, e.g. SLB, ENF
  - Relatively thick compared to specimen length; specimen dimension coupling
  - Limited space for crack to propagate
- We want “axial type” specimen to test crack arrest behavior
  - Symmetric, 3-beam model, load applied to the center beam



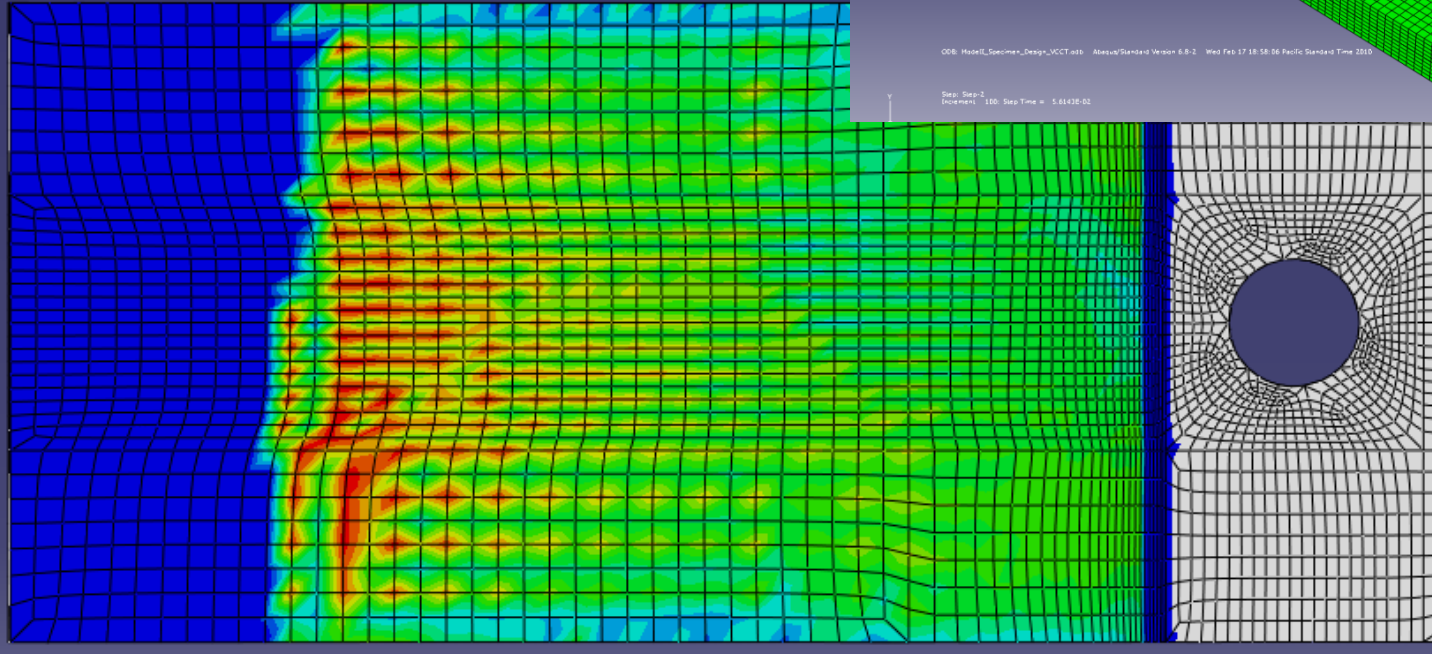
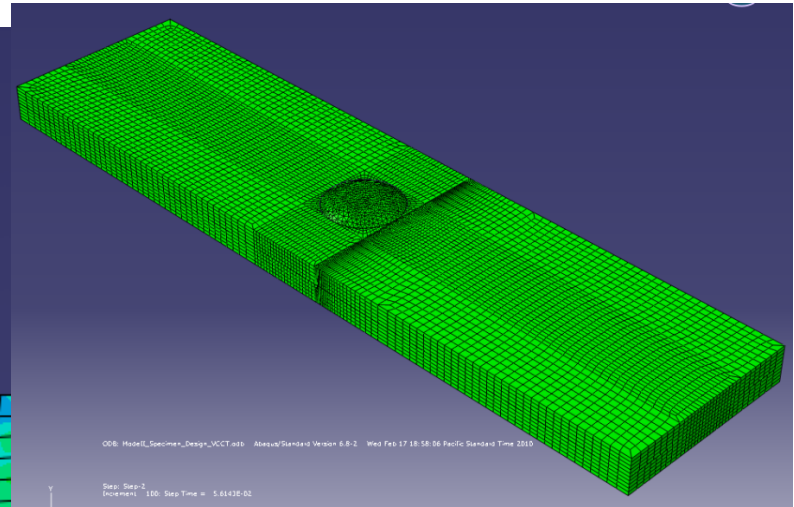
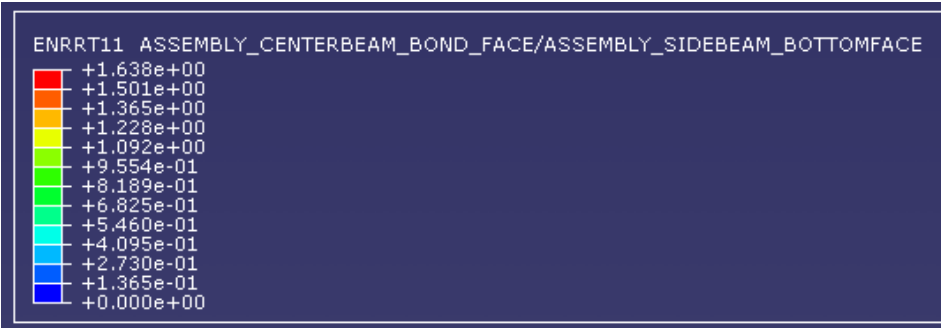
# Mode II Test Specimen Preliminary Findings



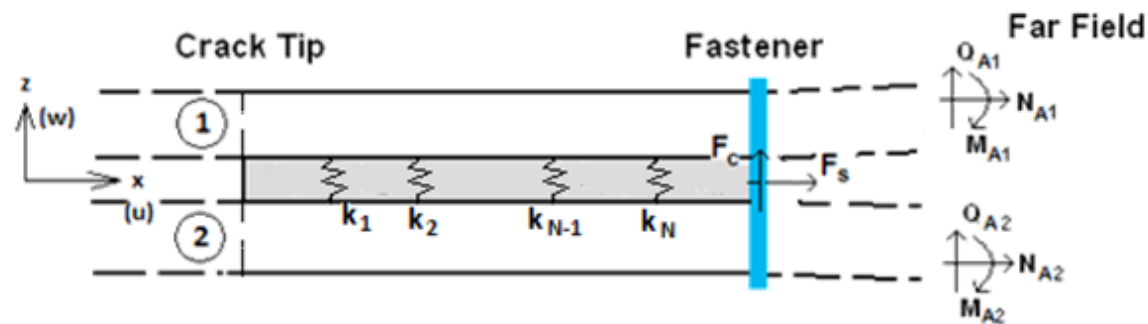
The Joint Advanced



# Mode II Test Specimen in 3-D



- Uses Rayleigh-Ritz method and the energy principle.
- Two beams, fastener (two springs), and an elastic foundation layer between beams.
- Elastic layer is composed of  $N$  individual springs where  $k$  is very large in compression and zero in tension, for contact and separation.
- Solve system for the state of minimum potential energy iteratively.
- SERR mode decomposition by Wang/Qiao.



$$\delta\Pi = 0; \quad \text{where } \Pi = U_{\text{total}} - W_{\text{total}}$$

## Beam Energy Terms

$$U_b = \frac{1}{2}EI \int_0^L \left( \frac{d^2w}{dx^2} \right)^2 dx$$

$$U_s = 1.2 \frac{EI^2}{A} (1 + \nu) \int_0^L \left( \frac{d^3w}{dx^3} \right)^2 dx$$

$$U_{ba} = \frac{1}{2}N \int_0^L \left( \frac{dw}{dx} \right)^2 dx$$

## Elastic Layer Energy

$$U_{EL} = \sum_{n=1}^N \frac{1}{2} k_n (w_2 - w_1)^2 \Big|_{x=L}^{(n)}$$

## Fastener/Spring Energy

$$U_{kc} = \frac{1}{2} k_c (w_2 - w_1)^2 \Big|_{x=L}$$

$$U_{ks} = \frac{1}{2} k_s (u_2 - u_1)^2 \Big|_{x=L}$$

## Work Terms

$$W_Q = Qw \Big|_{x=L}$$

$$W_M = M \left( \frac{dw}{dx} \right) \Big|_{x=L}$$



- Shape functions must satisfy geometric boundary conditions.
  - Shape functions considered for transverse displacement.

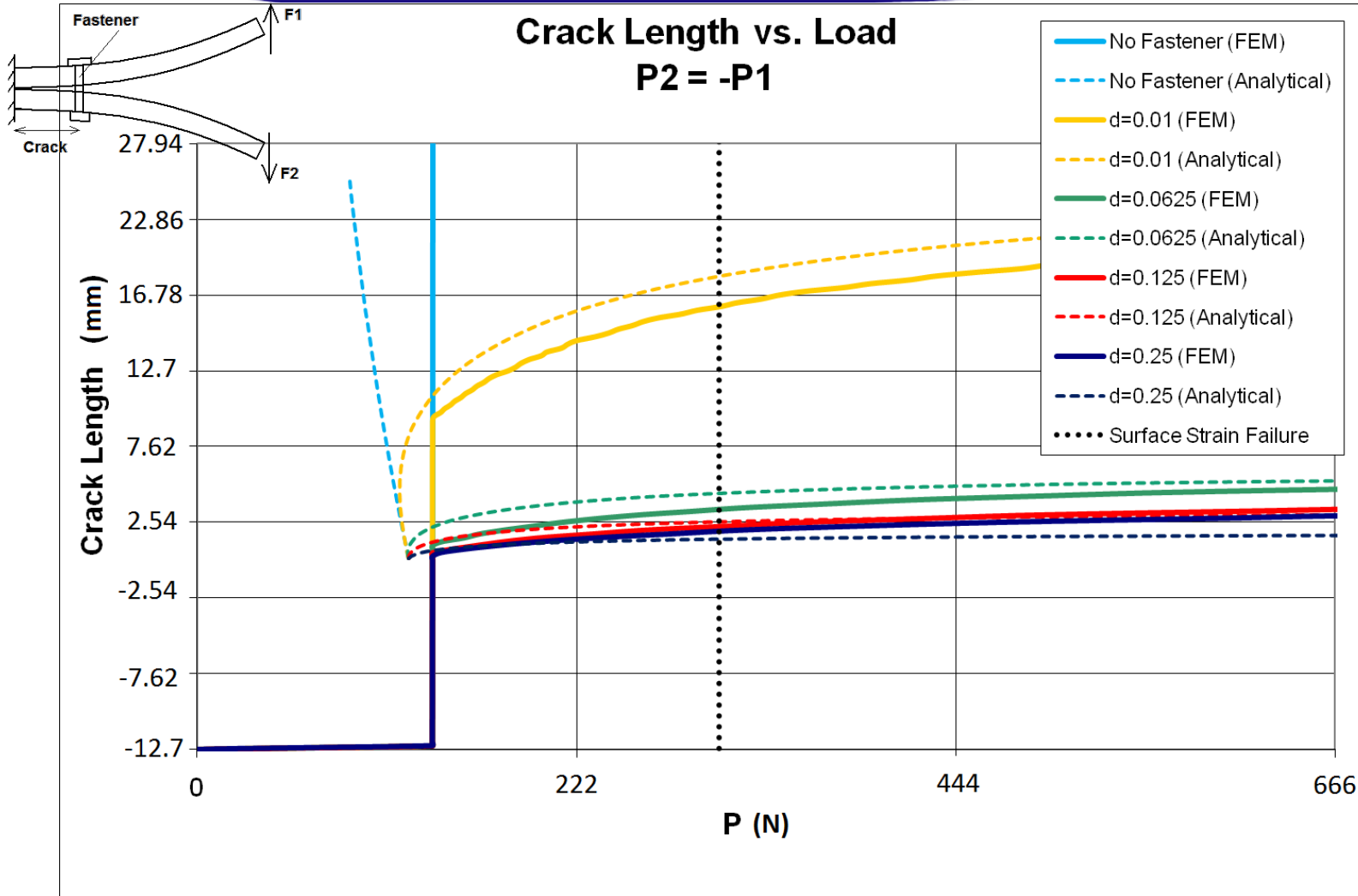
$$w_1 = \sum_{i=1}^I \alpha_i x^{i+1} ; i = 1, 2 \dots I$$

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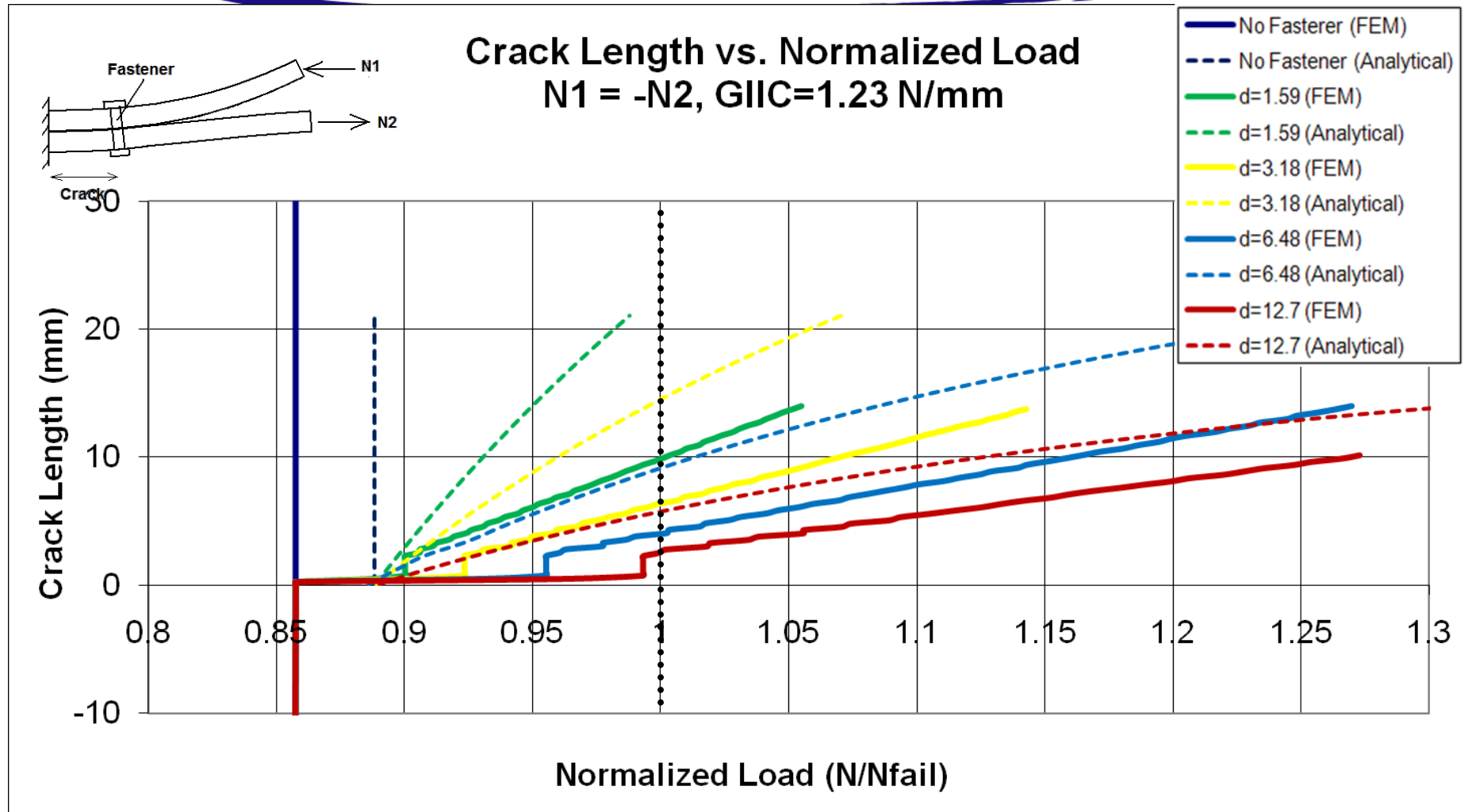
- Shape functions considered for axial displacement

$$u_1 = x \left( 1 + \frac{N_1}{A_1 E_1} \right) \qquad u_2 = x \left( 1 + \frac{N_2}{A_2 E_2} \right)$$

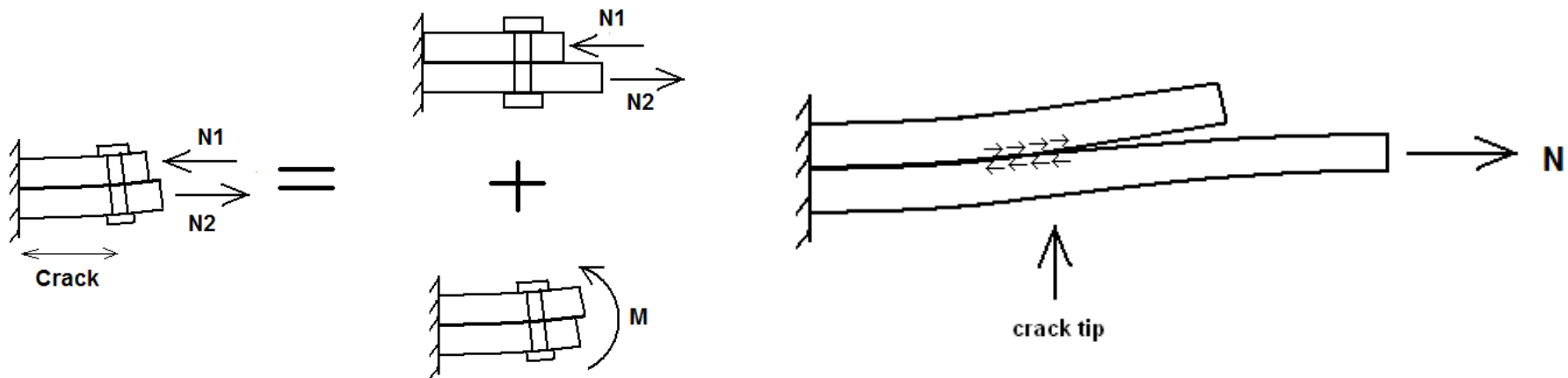
# Mode I: FEM vs. Analytical Fracture Analysis



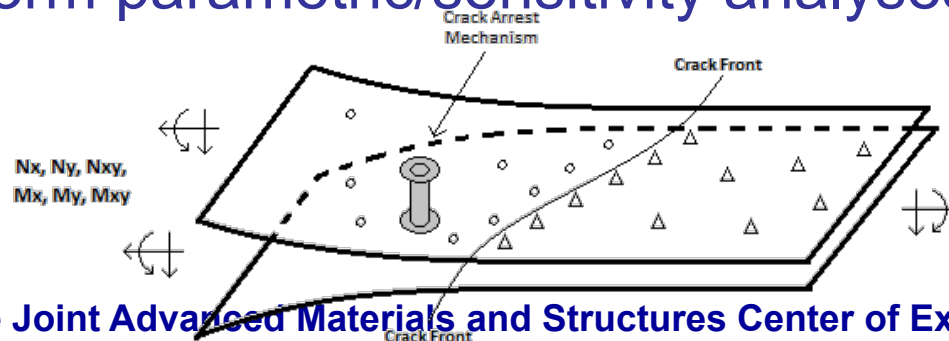
# Mode II: FEM vs. Analytical Fracture Analysis



- Opposite axial forces causes a bending moment that counteracts the relative displacement of the beams at the location of the fastener.
- Traction acting on intact portion of the beams contributes to moment coupling through equilibrium requirements.
- Use piece-wise adaptive shape functions to model the entire beam



- Develop analytical solutions
- Consider all alternate failure modes
- Model crack propagation around the fastener in 3-D
- Consider multiple fasteners
- Design validation experiments
- Generate design curves
- Identify key variables for design and optimization
- Perform parametric/sensitivity analyses



## ▪ **Benefit to Aviation**

- Provide analysis tools for fastener arrest mechanism
- Provide a fail-safe path to the design of integrated composite structures
- Optimization can lead to weight savings while properly addressing safety issue
- Integrating with probabilistic analysis method can properly address design uncertainties

# [back up] Model Description

- 16-ply CFRP (  $t = 0.0075'' \times 16 = 0.12''$  )
- Lay-ups
  - Percentage of 0-deg: 25% / 37.5% / 50% / 62.5%
- Fastener
  - Ti-Al6-V4 (  $E = 16.5 \times 10^6 \text{psi}$  )
  - $d = 0.25 \text{ in}$
- Fastener Flexibility (H. Huth, 1986)

$$C = \left( \frac{t_1 + t_2}{2d} \right)^a \frac{b}{n} \left( \frac{1}{t_1 E_1} + \frac{1}{nt_2 E_2} + \frac{1}{nt_1 E_3} + \frac{1}{2nt_2 E_3} \right)$$

# [back up]

## Material Properties (AS4/3501-6)

- $E_1=127.5\text{GPa}$
- $E_2=11.3\text{GPa}$
- $G_{12}=6.0\text{GPa}$
- $\nu=0.3$
- $X_t=2282\text{MPa}$
- $X_c=1440\text{MPa}$
- $Y_t=57\text{MPa}$
- $Y_c=228\text{MPa}$
- $S_{xy}=71\text{MPa}$
- $G_{IC}=0.2627\text{N/mm}$
- $G_{IIC}=1.226\text{N/mm}$
- $\eta=1.75$
- $E_1=18.5\text{Msi}$
- $E_2=1.64\text{Msi}$
- $G_{12}=0.871\text{Msi}$
- $\nu=0.3$
- $X_t=331\text{ksi}$
- $X_c=208.9\text{ksi}$
- $Y_t=8.3\text{ksi}$
- $Y_c=33.1\text{ksi}$
- $S_{xy}=10.3\text{ksi}$
- $G_{IC}=1.5\text{lb/in}$
- $G_{IIC}=7.0\text{lb/in}$
- $\eta=1.75$