

# CRACK DEVELOPMENT IN CYCLICALLY LOADED PRESSURIZED CYCLINDRICAL CARBON FIBER SHELL STRUCTURES

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# OBJECTIVE

- Determine the contribution of the bulging effect in crack development for Fiber-Polymer Composites (FPCs).
- Determine the relationship between fiber orientation and the bulging effect on FPCs.
- Present a sound bench scale test that can be used to study the bulging phenomenon.
- The ultimate goal is to provide engineers with a relationship between measurable changes in geometry around the crack lip and damage evolution on FPCs.

# Failure Mechanisms Involved In Crack Development

Ductile Isotropic materials (Clean failure):

1. Crack initiation
2. Crack propagation
3. Sudden fracture



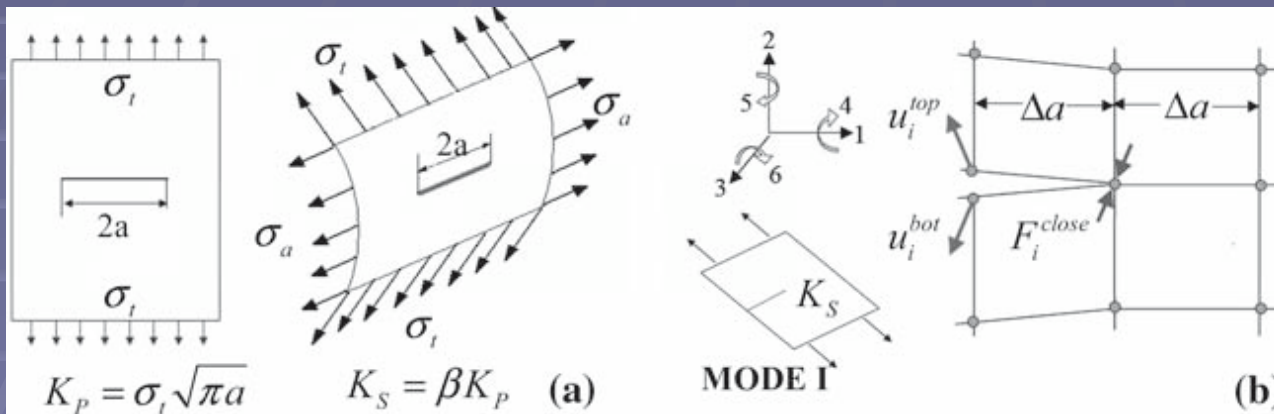
Fiber-Polymer Composites (Dirty failure):

1. Crack initiation
2. Edge crack increase without inward propagation
3. Transverse crack propagation



# Concept of Bulging Factor

Bulging is the consequence of the presence of a crack on a pressurized cylindrical shell structure that leads to complex stress and displacements fields resulting in nonlinear out-of-plane deformations.



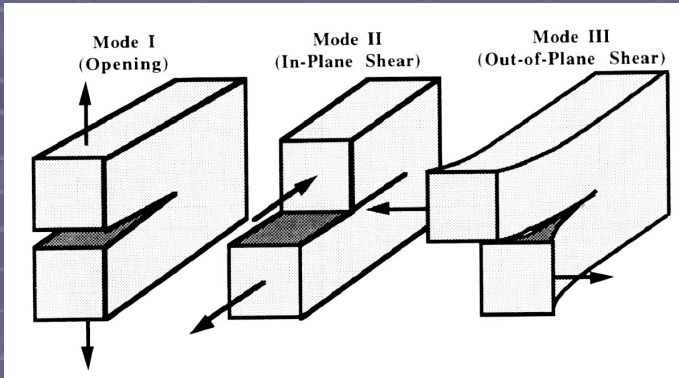
$$\beta = \frac{K_{1curved}}{K_{1flat}}$$

$$K_{1flat} = \frac{pR}{t} \sqrt{\pi a}$$

$$\chi = \frac{\sigma_{long}}{\sigma_{hoop}}$$

$$\beta = \sqrt{1 + \frac{5}{3\pi} \frac{Eta}{R^2 p} \frac{0.316}{\sqrt{1+18\chi}} \tanh \left( 0.06 \frac{R}{t} \sqrt{\frac{pa}{Et}} \right)}$$

# 3-D Modified Crack Closure Integral

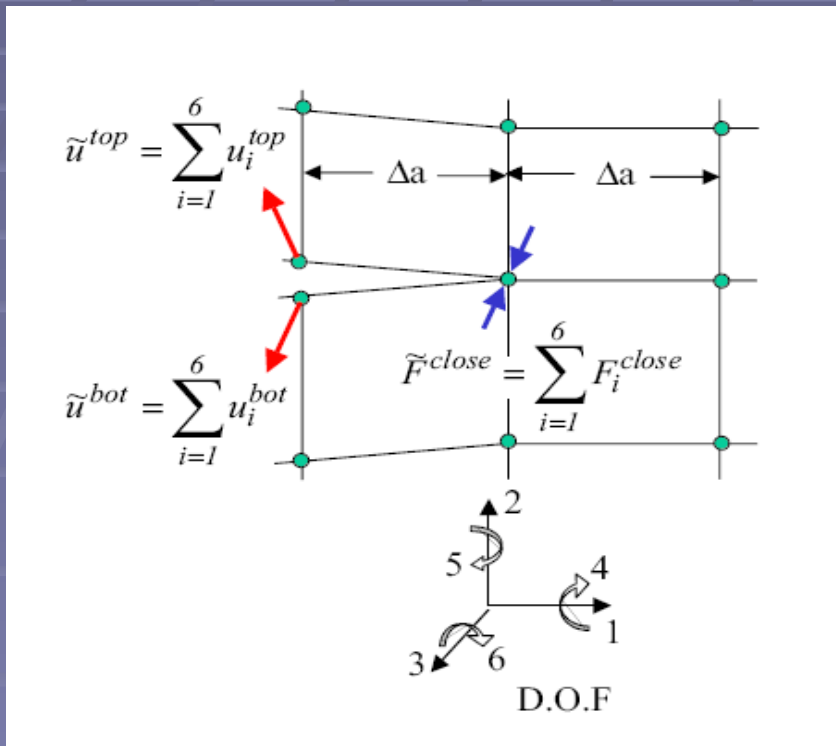


$$u_x = \frac{K_I}{2\mu} \sqrt{\frac{r}{2\pi}} \cos\left(\frac{\theta}{2}\right) \left[ \kappa - 1 + 2 \sin^2\left(\frac{\theta}{2}\right) \right]$$

$$u_y = \frac{K_I}{2\mu} \sqrt{\frac{r}{2\pi}} \sin\left(\frac{\theta}{2}\right) \left[ \kappa - 1 + 2 \cos^2\left(\frac{\theta}{2}\right) \right]$$

$$\mu = \frac{E}{2(\nu + 1)}$$

$$\kappa = (3 - \nu) / (1 - \nu)$$



$$W_i = \frac{1}{2t\Delta a} \left[ F_i^{close} \left( u_i^{top} - u_i^{bottom} \right) \right]$$

$$W_2 + W_6 = \frac{K_1^2}{E}$$

$$W_1 = \frac{K_2^2}{E}$$

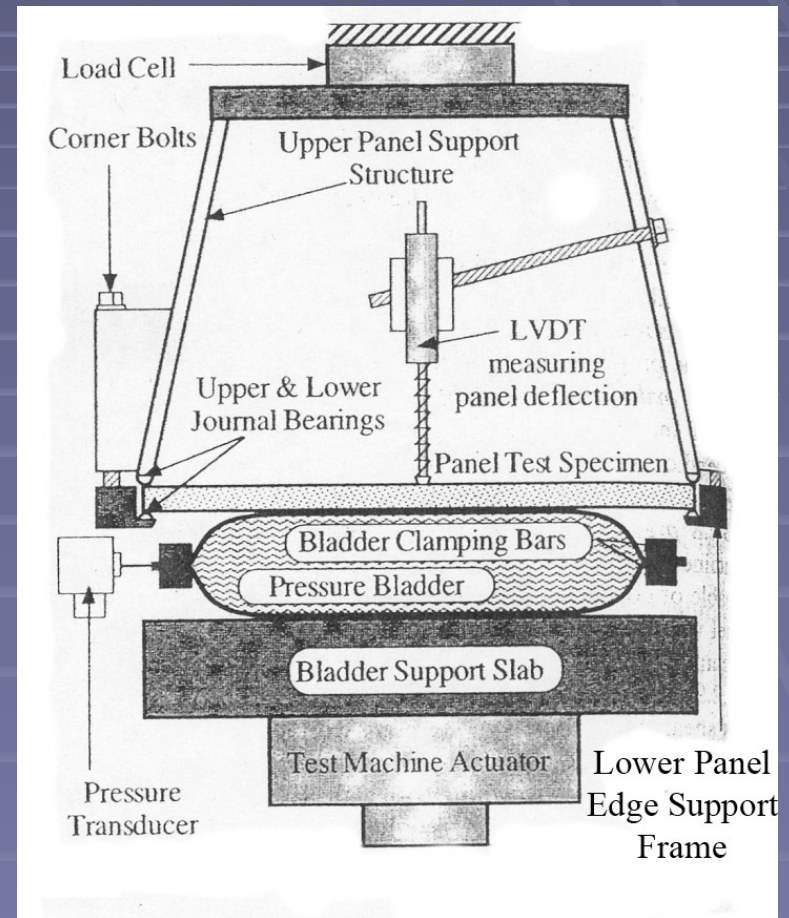
$$W_4 = \frac{k_1^2 \pi}{3E} \left( \frac{1 + \nu}{3 + \nu} \right)$$

$$W_3 + W_5 = \frac{k_2^2}{3E} \left( \frac{1 + \nu}{3 + \nu} \right)$$

# Test Infrastructure

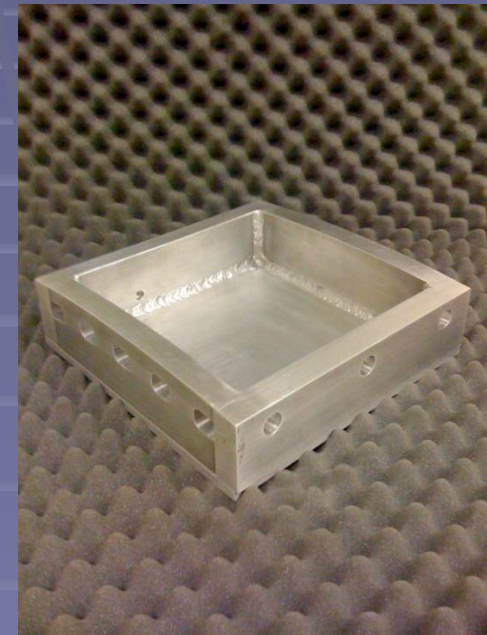
ASTM D 6416/D 6416: This standard was designed to determine the two-dimensional flexural properties of sandwich composite plates under a distributed load. This standard consists of a square specimen, steel clamping frame and pressure bladder.

All these elements are mounted on an Universal Testing Machine (UTM). The loading fixture is made up of a two-piece rigid part that is bolted to the crosshead that is then fixed at the top and is pressed against a bladder filled-up with fluid to provide an evenly distributed pressure.



# Experimental Test Bed

The pressure chamber was designed to accommodate a 12" X 12" specimen having the same curvature as a hatch service door from a Boeing 727. One of the major challenges was to replicate the curvature of the part accurately. This factor is extremely important since it is the basis for the bulging phenomenon.

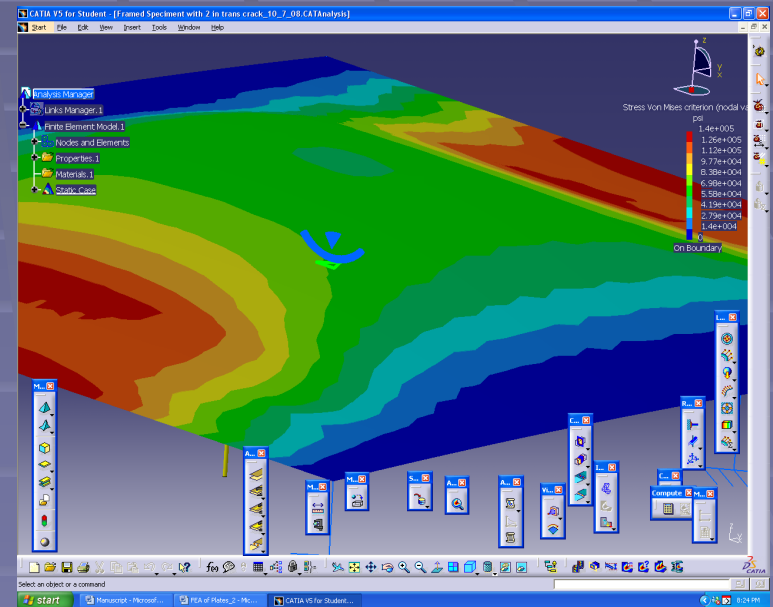
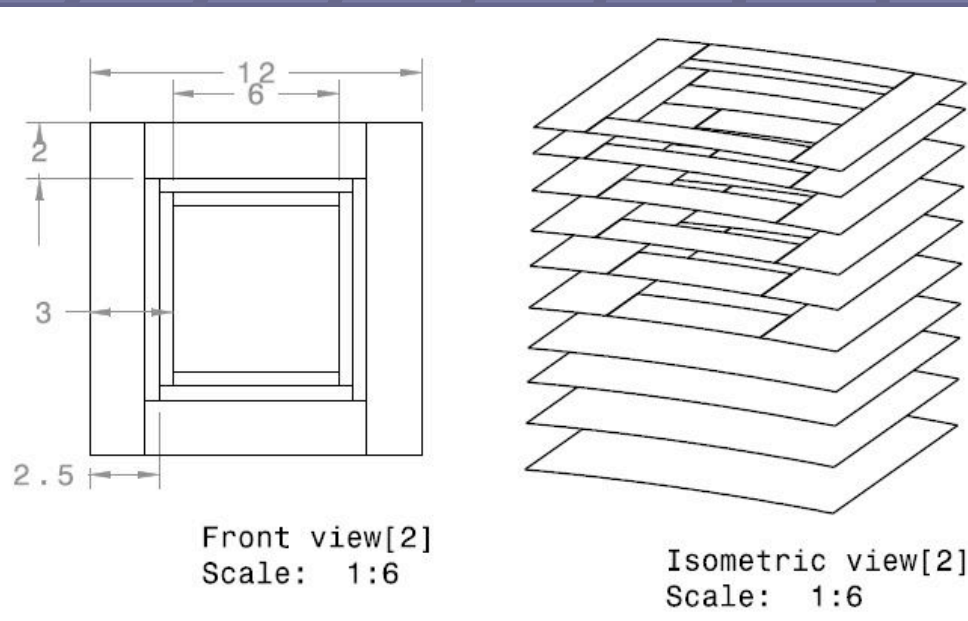






# Proposed Specimen Design

In order to drive failure areas away from the edges, a gradual increase of the number of plies is staggered toward the critical zones



# Validation of Numerical Methodology (AI Curved Shell)

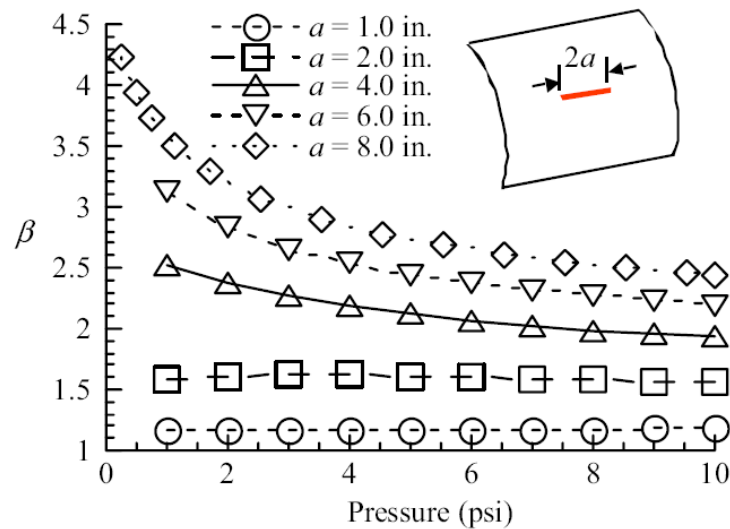


TABLE 3. KIRCHOFF SIF AND  $\beta$  FOR A LONGITUDINAL CRACK IN AN UNSTIFFENED FUSELAGE WITH  $R = 80$  in. AND  $a = 2.0$  in.

$p$ (psi)	$K_{1\ flat}$ (psi $\cdot$ $\sqrt{\text{in}}$ )	$K_1$ (psi $\cdot$ $\sqrt{\text{in}}$ )	$K_2$ (psi $\cdot$ $\sqrt{\text{in}}$ )	$k_1$ (psi $\cdot$ $\sqrt{\text{in}}$ )	$k_2$ (psi $\cdot$ $\sqrt{\text{in}}$ )	$\beta$
1.00	4.178e+03	6.583e+03	0	1.874e+03	0	1.576
2.00	8.355e+03	1.342e+04	0	3.917e+03	0	1.607
3.00	1.253e+04	2.030e+04	0	5.721e+03	0	1.620
4.00	1.671e+04	2.704e+04	0	7.190e+03	0	1.618
5.00	2.089e+04	3.364e+04	0	8.395e+03	0	1.610
6.00	2.507e+04	4.010e+04	0	9.413e+03	0	1.600
7.00	2.924e+04	4.645e+04	0	1.029e+04	0	1.588
8.00	3.342e+04	5.271e+04	0	1.107e+04	0	1.577
9.00	3.760e+04	5.889e+04	0	1.178e+04	0	1.566
10.00	4.178e+04	6.500e+04	0	1.242e+04	0	1.556

Pressure	Calculated B.F.	B.F. Chen	B.F. FAA	% Error (Chen)
2	1.792	1.758	1.61E+00	1.9
4	1.792	1.563	1.62E+00	12.8
6	1.792	1.466	1.60E+00	18.2
8	1.792	1.405	1.58E+00	21.6

# Calculation of Bulging Factor on Composites Laminates

Composite  
[0/90/0]

R = 120

EI = 1.32E+07

Et = 7.33E+06

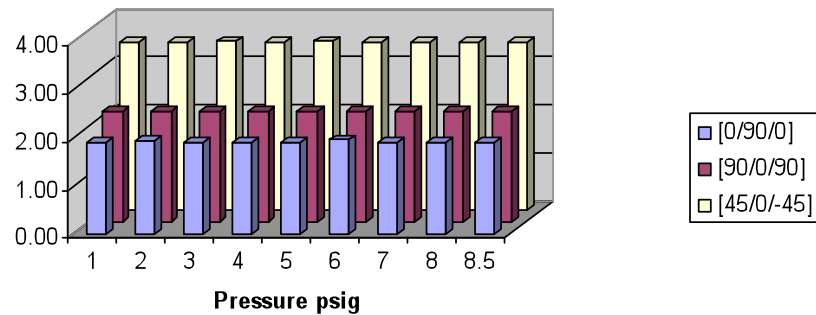
t = 0.018

Pressure	Force	Moment	Displ	W2	W6	KI (Long)	KI (Hoop)
1	47.81	5.89	0.00071	125.723	15.489	4.312E+04	3.217E+04
2	95.62	11.78	0.00145	513.515	63.263	8.716E+04	6.502E+04
3	143.42	17.67	0.00213	1131.424	139.397	1.294E+05	9.651E+04
4	191.23	23.56	0.00284	2011.456	247.816	1.725E+05	1.287E+05
5	239.04	29.45	0.00355	3142.933	387.213	2.156E+05	1.609E+05
6	286.85	35.34	0.00448	4759.585	586.382	2.653E+05	1.980E+05
7	334.66	41.23	0.00497	6160.223	758.937	3.019E+05	2.252E+05
8	382.46	47.12	0.00568	8045.825	991.265	3.450E+05	2.574E+05
8.5	406.37	50.06	0.00603	9075.597	1118.007	3.664E+05	2.733E+05

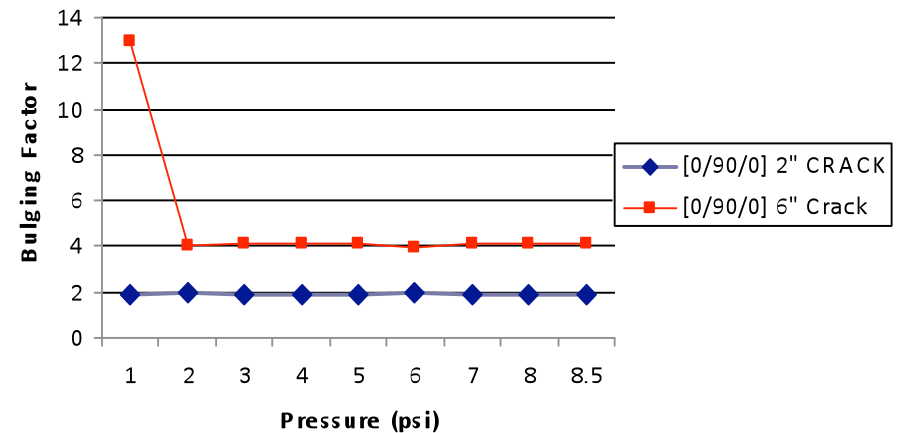
Pressure	K Flat	B.F.	B Chen
1	1.67E+04	1.925	1.752
2	3.34E+04	1.945	1.508
3	5.01E+04	1.925	1.388
4	6.68E+04	1.925	1.314
5	8.36E+04	1.925	1.264
6	1.00E+05	1.974	1.228
7	1.17E+05	1.925	1.200
8	1.34E+05	1.925	1.178
8.5	1.42E+05	1.924	1.169

# Calculation of Bulging Factor on Composites Laminates

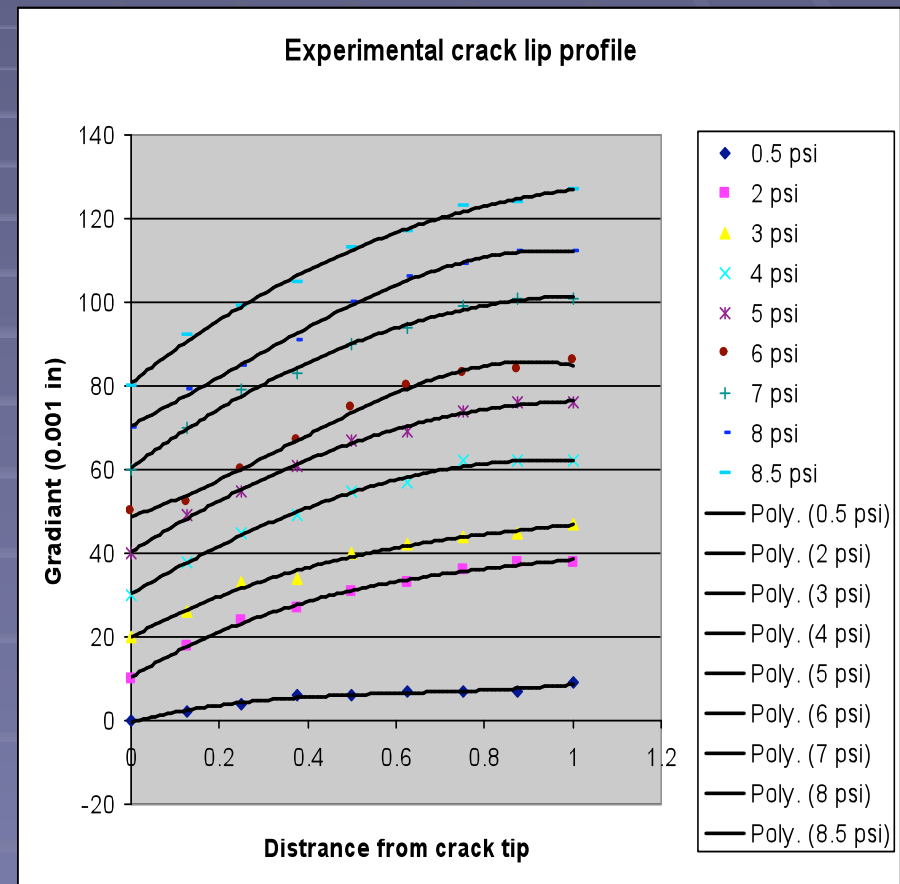
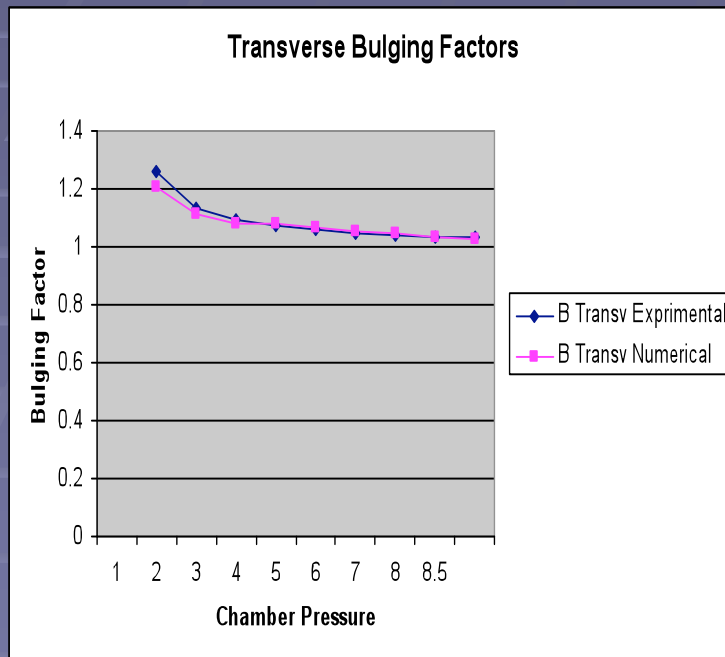
Bulging Factors of Different Ply Configurations - Radius 120"



Bulging Factor vs. Pressure - 0/90/0



# Calculation of Experimental Bulging Factor on Composites Laminates



# Conclusions

- For a two inch crack, the bulging factor was found to be independent of pressure.
- For FPCs, the stiffer the panel, the higher the bulging factor.
- MCCI method can be used to accurately determine stress intensity factors for FPCs.
- A small scale test bed was created that shows promise of measuring bulging factor characteristics.
- It was shown that significant accuracy can be obtained by including collapsed elements in the FE analysis.

# Future Work

- The small specimen size utilized in this study resulted in a small variation in crack size (0.001") during cycling. Larger specimens would provide for more accurate test measurements required for the MCCI method.
- These tests were conducted with panels without support. Larger panels with stringers would provide a more realistic representation of the aircraft fuselage.
- Characterize the failure stages for curved FPCs under pressurization/depressurization cyclic loading conditions.
- Optimize FEA results by using software that will include interlaminar shear stresses.